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**Backpressure Prediction for Flow Through Monoliths  
and Wall Flow Filters Using 1 Dimensional Models:  
Entrance Effect Pressure Change, Developing Flow and Validation  
Using Length Varying Techniques**

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# Objective

## Open Questions on Backpressure Prediction for Monoliths and Particulate Filters

- Aim to address a number of open questions on backpressure prediction for Flow-Through Monoliths (FTM) and Particulate Filters (PF) using 1-D models:
  - No agreement on equation for pressure drop due to flow contraction at FTM or PF entrance
  - How should friction factor correlations for developing flow in FTM be combined with an equation for flow contraction pressure loss to predict backpressure across an FTM?
  - How to discriminate between different PF models from literature?
  - Should developing flow be included in PF models?

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# Methods / Techniques

## How are we going to answer the questions

- Use three methods:
  - 1) CFD simulations
  - 2) Backpressure measurements on monolith cut to different lengths
  - 3) Backpressure measurements on particulate filters cut to different lengths
- Varying length of part helps with separating different contributions to backpressure
  - Although more complicated with filters than monolith

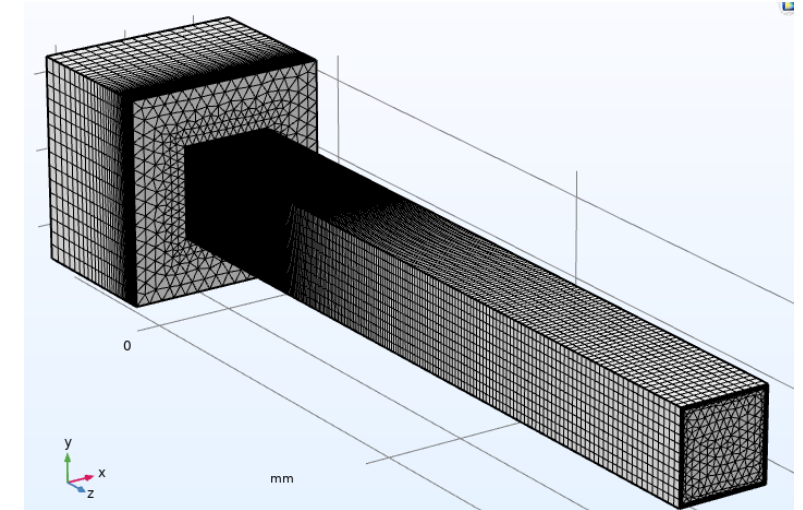


CFD Simulations

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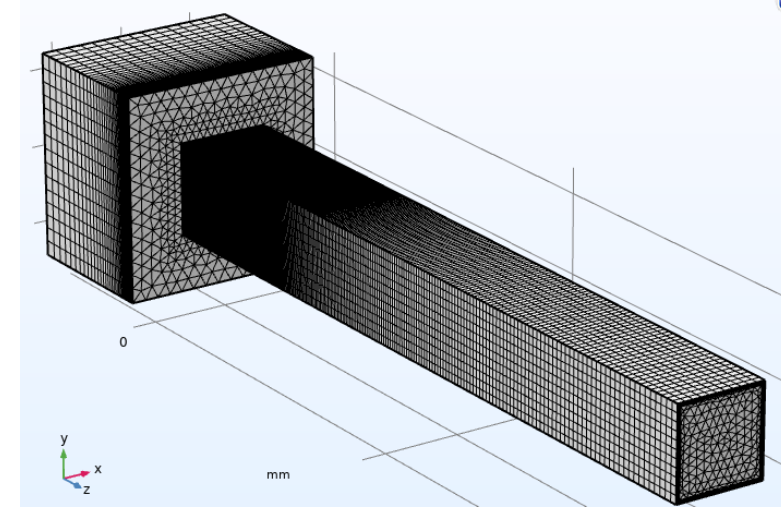
# CFD Simulations

- Simulate flow in square channel with upstream flow contraction
  - Model for flow encountered in flow-through monolith
  - Investigate the effect of:
    - Gas velocity
    - Contraction ratio
- Enables us to:
  - Verify pressure change due to flow contraction at part entrance
  - Look at the impact of developing flow
  - Consider how upstream contraction affects developing flow in channel
    - Friction factor correlations typically assume flat velocity profile at channel entrance
    - But this does not exist when there is an upstream flow contraction



# Simulations

- CFD simulations run using Comsol Multiphysics®
  - Geometry consists of square channel with upstream contraction
  - Simulations run for:
    - Two flow rates
    - 4 contraction ratios, plus the case without an upstream contraction



## Simulation Geometry:

$$\sigma = \frac{\text{Area for flow after contraction}}{\text{Area for flow before contraction}}$$

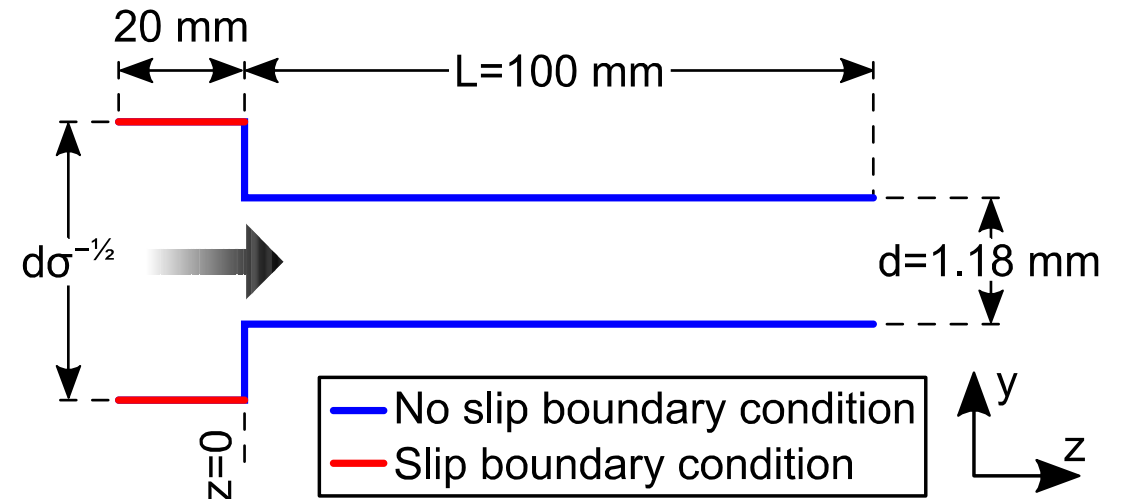


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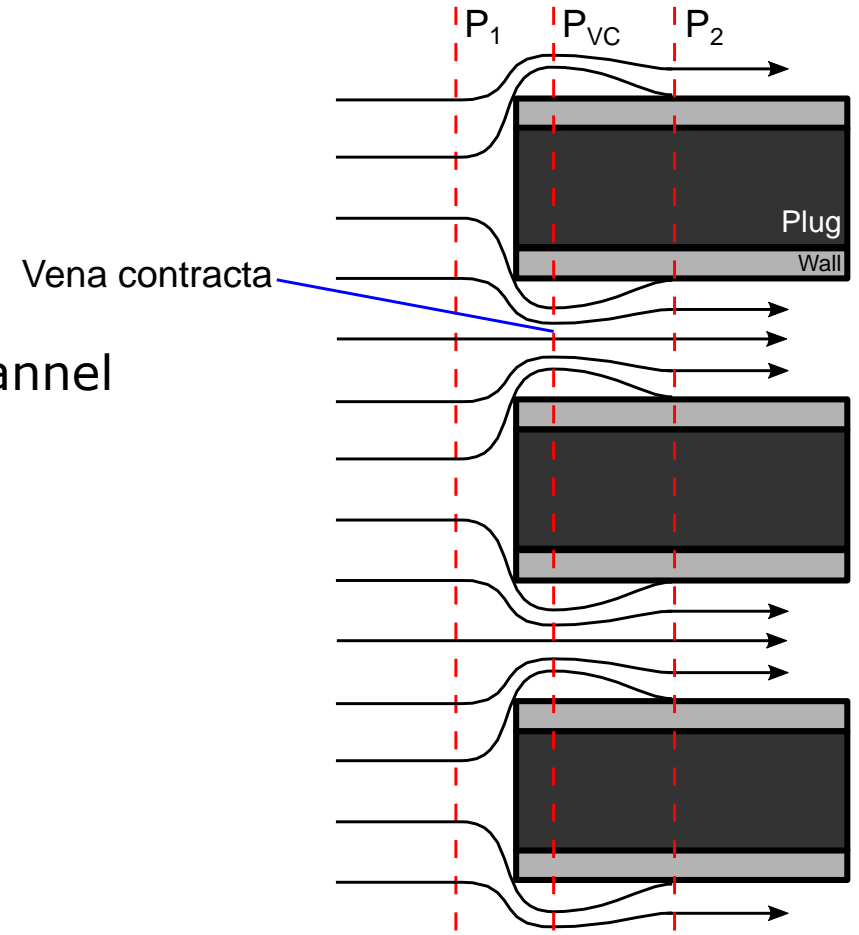
# Pressure Drop Due to Flow Contraction

- During flow contraction at monolith/filter entrance:
  - Flow contracts down to vena contracta, narrower than channel
  - Then flow expands to fill channel
- Derive equation for pressure change assuming [2,5]:
  - Conservation of mechanical energy to vena contracta
  - Conservation of momentum over expansion

$$\Delta P_{entr} = \left[ 2\alpha_{i0} - \alpha_{e,up}\sigma_{i0}^2 - \frac{2}{C_{ci}} + \frac{1}{C_{ci}^2} \right] \frac{\rho V_{i0}^2}{2}$$

- $\alpha_{i0}$  is momentum flux correction factor
  - Serves to correct  $\rho V^2$  to give momentum flux along channel

$$\alpha = \frac{\langle u_z |u_z| \rangle}{\langle u_z \rangle^2}$$



$C_C$  = Coefficient of contraction  
 $V$  = Mean channel velocity  
 $\alpha$  = Momentum flux correction factor  
 $\alpha_e$  = Kinetic energy flux correction  
 $\rho$  = Fluid density  
 $\sigma_{i0}$  = Contraction ratio

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# Handling Backpressure Contribution Due to Accelerating Gas Flow to Laminar Flow Profile

- One contribution to backpressure for gas entering channel is force required to accelerate gas flow in channel to laminar flow profile
- If:
  - 1) Assume flow in channel always fully developed
    - Include this contribution with contraction pressure loss by using  $\alpha_{i0}$  for laminar flow
  - 2) Account for developing flow in channel
    - Use friction factor correlation that accounts for developing flow
      - This includes contribution for force required to accelerate gas to laminar flow profile
    - Equation for flow contraction should just take gas to flat velocity profile, i.e.  $\alpha_{i0} = 1$



# Some Maths...

## Dimensionless pressure and axial coordinate

- Pressure drop at distance  $z$  along channel,  $P_z$ , is given by:

- $$P_{up} - P_z = \left( \zeta_0^F + f_{app} \frac{4z}{d} \right) \frac{\rho V^2}{2}$$

- Divide by  $\frac{1}{2}\rho V^2$  and rearrange gives:

- $$P_z^* - P_{up}^* + \zeta_0^F = -4f_{app} Re z^+$$

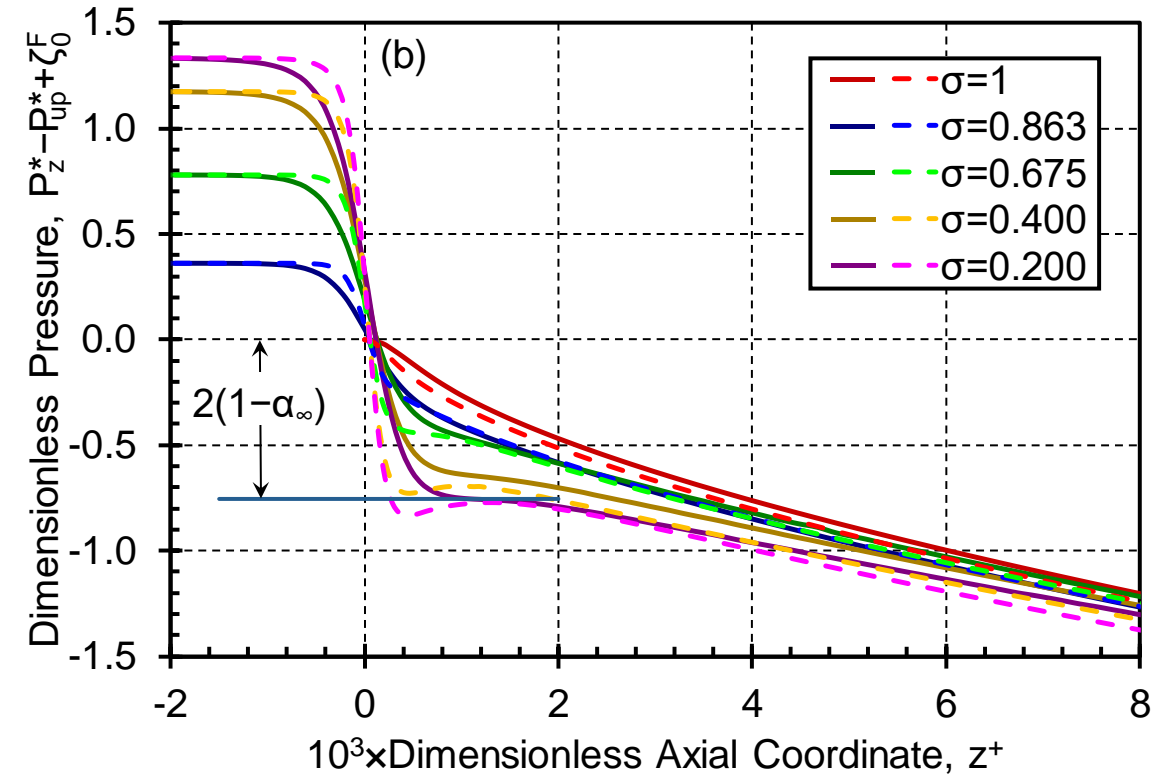
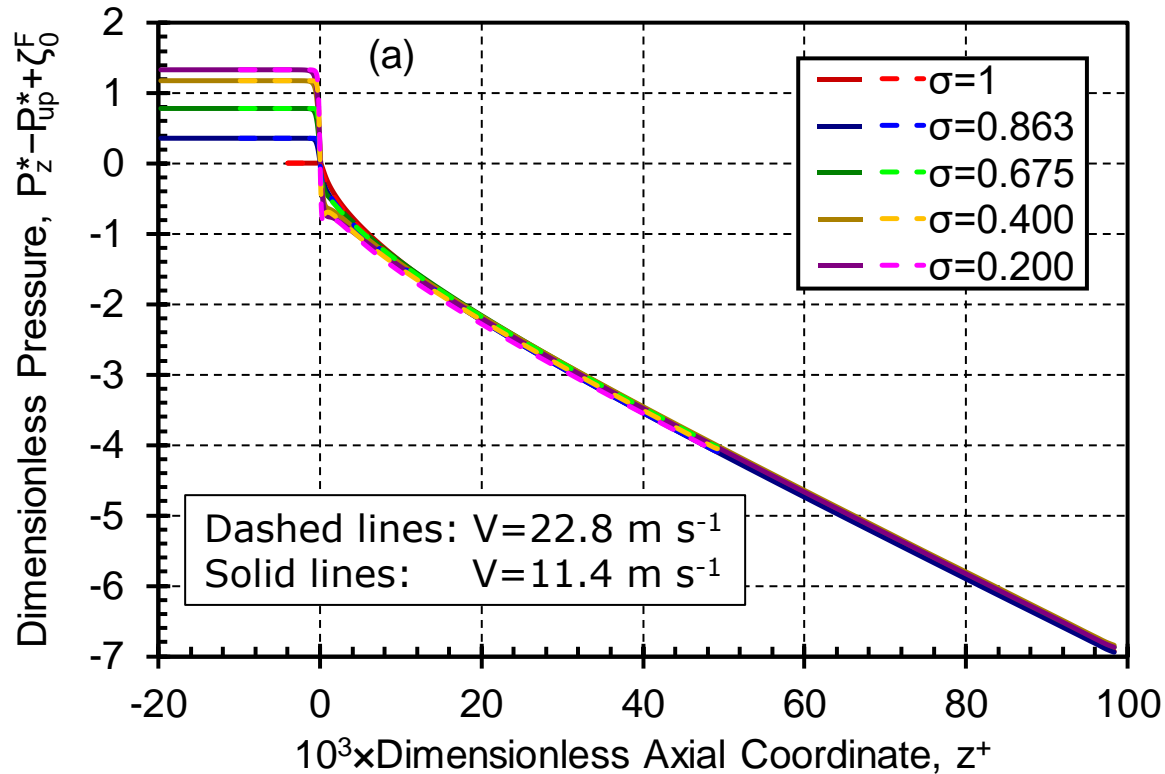
- From this equation, if  $f_{app} Re$  is a function of  $z^+$  only, then a plot of  $P_z^* - P_{up}^* + \zeta_0^F$  versus  $z^+$  will always fall on the same line for  $0 \leq z \leq L$  irrespective of contraction ratio or flow rate

- Not entirely true close to the contraction, but works otherwise

$P_{up}$ =upstream pressure;  $P_z$ =pressure at  $z$ ;  $z$ =axial coordinate;  $\zeta_0^F$ =pressure drop due to contraction, taking fluid to a flat velocity profile;  $f_{app}$ =apparent friction factor;  $d$ =width of channel;  $\rho$ =fluid density;  $V$ =mean gas velocity in channel;  $P^*=P/\frac{1}{2}\rho V^2$ =dimensionless pressure;  $Re=dpV/\mu$ =Reynolds number;  $z^+=z/dRe$ =dimensionless axial coordinate

# Offset Dimensionless Pressure Versus Dimensionless Coordinate

All results line up when plotted this way, once away from the contraction



- Differences in pressure profile around the channel entrance due to contraction
- But once away from the contraction, all results fall on same line

# Conclusions from Dimensionless Pressure Drop Plot

## Consequences for backpressure prediction in flow-through monoliths

- If nondimensionalise pressure and axial coordinate and offset by pressure change due to flow contraction, all results fall on same curve away from contraction
  - For 2 flow rates and 5 contraction ratios (including no contraction)
- From this conclude:
  - Equation we are using for pressure drop due to contraction is correct (equation validated 😊)
  - Can predict pressure drop by combining correlation for developing flow starting with a flat velocity profile with an equation for pressure drop for contraction taking the flow to a flat velocity profile
  - There is no need to include a viscous contribution to contraction pressure drop
    - At least for the (relatively high) flow rates considered
    - Unlike what Cornejo et al. found [3,4]
  - While contraction affects pressure close to contraction, overall pressure drop unaffected
    - Unlike what Cornejo et al. found [3,4]

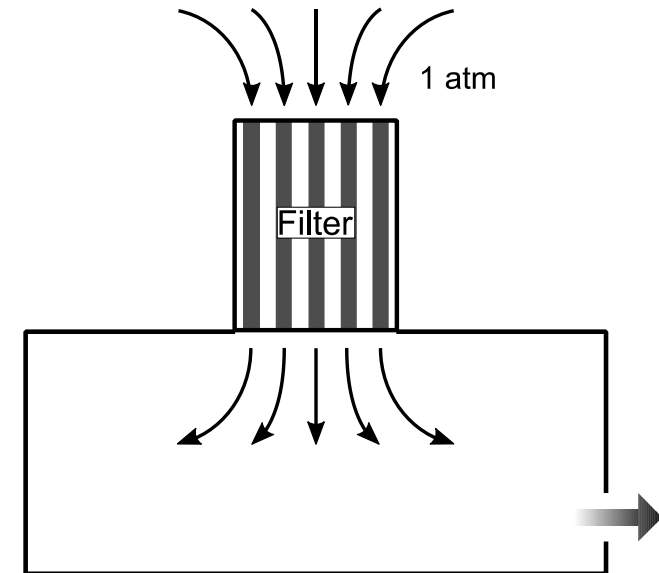


## Flow-Through Monolith Backpressure

# Backpressure Measurements on Flow-Through Monolith

## Effect of varying monolith length

- Measure backpressure of uncoated monolith, then progressively shorten part and re-measure backpressure
  - Gives backpressure as a function of length
    - Allows separation of along-channel pressure drop and pressure drop due to flow contraction into the part
    - As inlet pressure fixed (atmospheric) and exit pressure change zero (infinite expansion), equivalent to measuring pressure along channel
- Use SuperFlow<sup>®</sup> for measurements (cold flow)
- 400/3.5, 4.66 in diameter monolith



# Backpressure Prediction Equations

- Assume incompressible flow
- Along channel pressure drop:

- $$\Delta P_{FTM} = \frac{F \mu \phi L}{d^2 \rho}$$

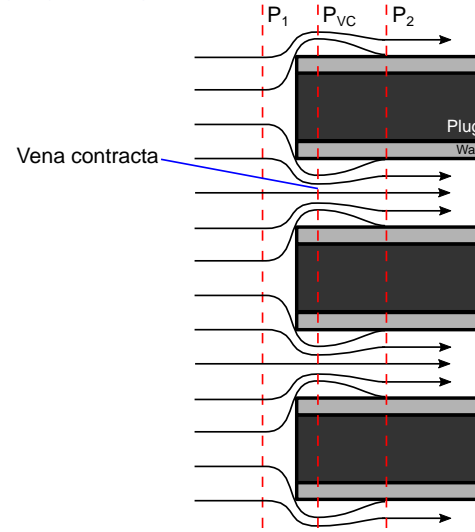
- Pressure drop due to flow contraction at entrance:

- $$\Delta P_{entr} = \left[ 2\alpha_{i0} - \alpha_{e,up} \sigma_{i0}^2 - \frac{2}{C_{ci}} + \frac{1}{C_{ci}^2} \right] \frac{\rho V_{i0}^2}{2}$$

- Depending on value of  $\alpha_{i0}$ , can take flow to flat profile or developed laminar flow profile

- Pressure change due to flow expansion at exit:

- $$\Delta P_{exit} = \sigma_{oL} (\alpha_{dwn} \sigma_{oL} - \alpha_{oL}) \rho V_{oL}^2$$



$d$ =Hydraulic diameter of channel  
 $F$ =Viscous loss coefficient  
 $L$ =length of monolith  
 $V$ =Mean gas velocity  
 $\alpha$ =Momentum flux correction factor  
 $\alpha_e$ =Kinetic energy flux correction  
 $\mu$ =viscosity  
 $\rho$ =Fluid density  
 $\sigma_{i0}$ =Contraction ratio  
 $\sigma_{oL}$ =Expansion ratio  
 $\phi$ =Mean mass flux in channel

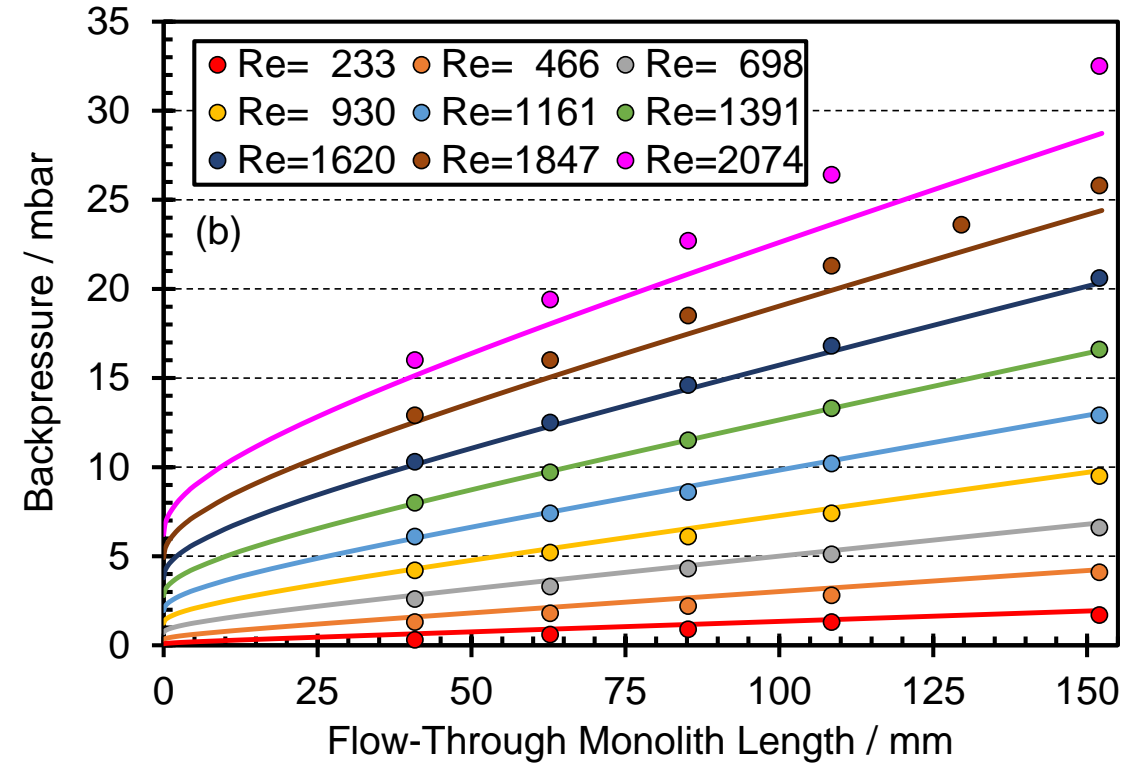
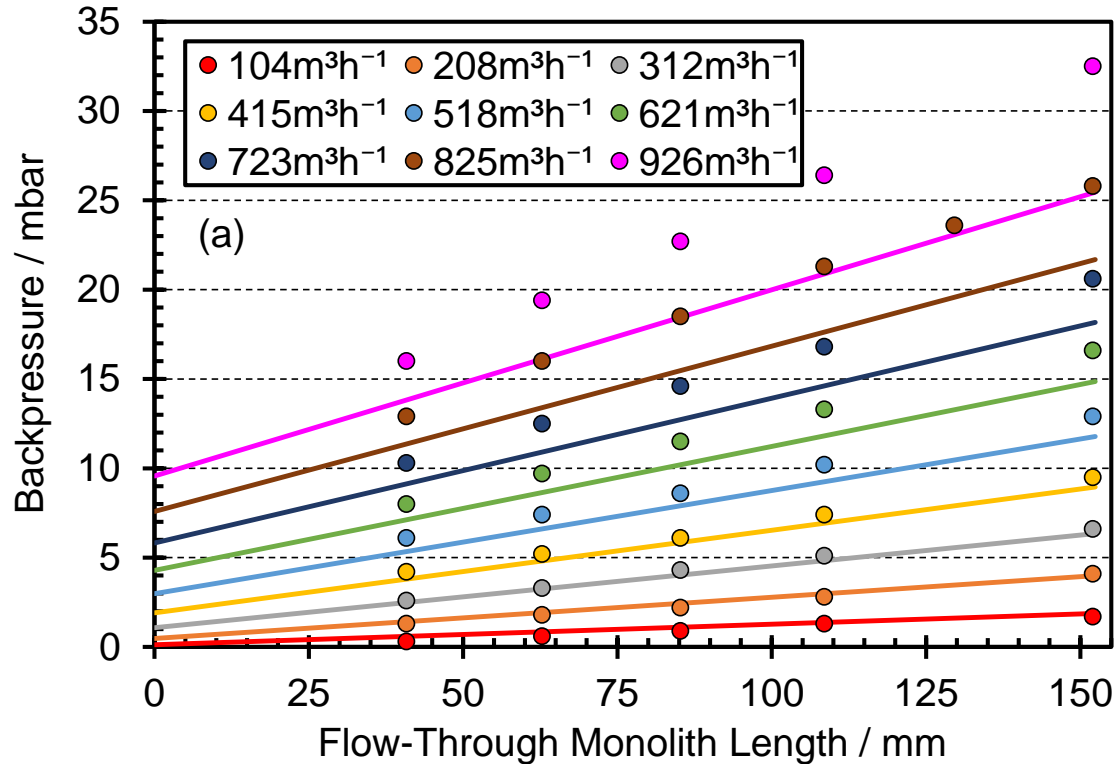
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Contraction & expansion equations from [2,5]

# Effect of Length on Monolith Backpressure

Points: Measured data  
Lines: Model prediction

Model predictions for always-developed flow (left) & accounting for developing flow (right)



- Validates equation for flow contraction into monolith
- At lower flow rates ( $\text{Re} \leq 700$ ), can assume flow always fully developed (left plot)
- Use of Gundlapally & Balokotiah [6] developing flow correlation improves prediction at higher flow
- Model under predicts at very high flows (right plot)

# Summary: Cut-Monolith Backpressure Data

- Backpressure of monolith measured as function of length by progressively shortening the part
- Allows backpressure contributions of along-channel pressure drop and entrance & exit effects to be separated
  - Improved model validation
- At low flow rates, can assume flow always developed, but at higher flows need to include the effect of developing flow
- Gundlapally & Balakotaiah friction factor correlation for developing flow works well

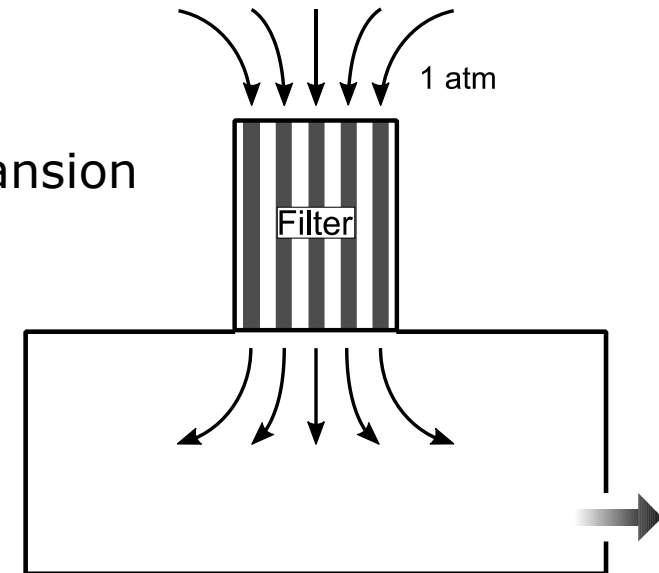




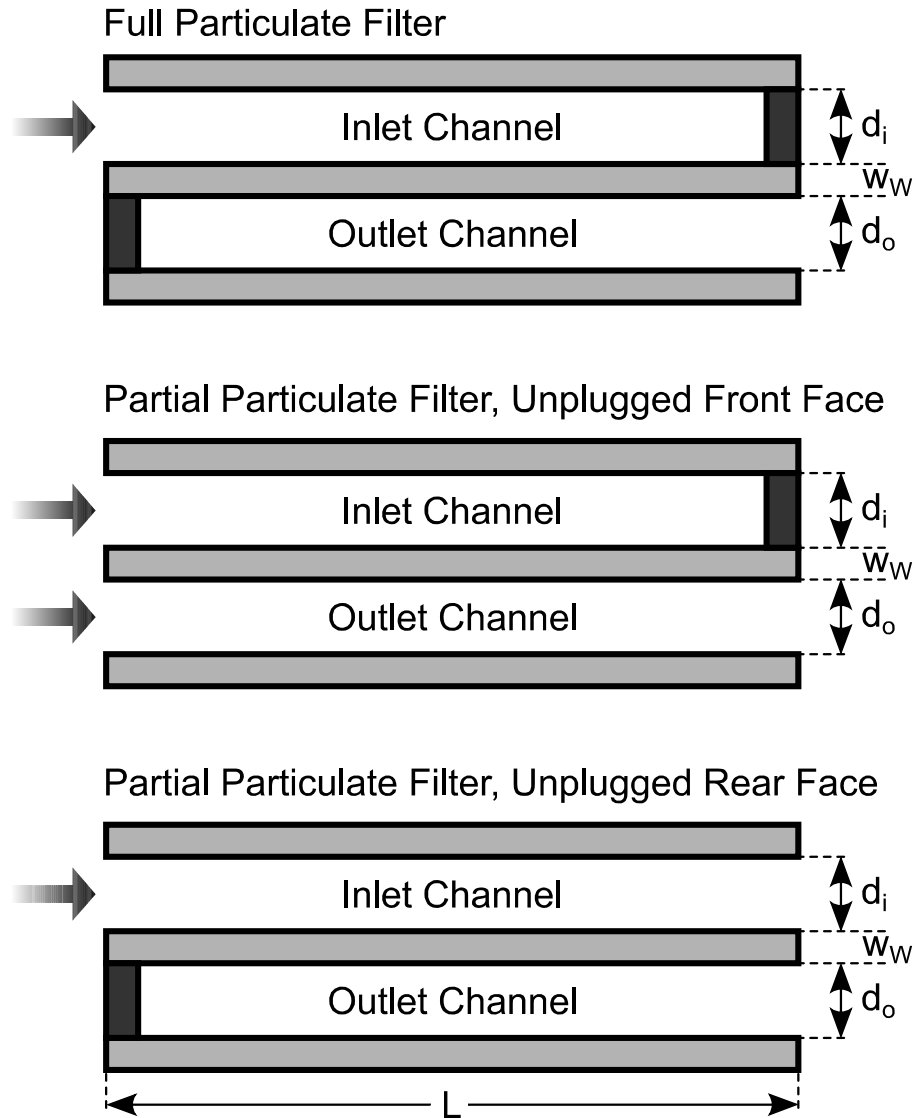
## Particulate Filter Backpressure Prediction

# Validation of Particulate Filter Backpressure Prediction

- Measure backpressure of uncoated filter, then progressively shorten part and re-measure backpressure
  - After first cut have partial filter – by testing in both orientations double available data
- Full and partial filters are governed by the same differential equations
  - Just with different boundary conditions
  - So can use data also to validate model for full filter
- Use SuperFlow<sup>®</sup> for measurements (cold flow)
  - Expansion pressure change zero as effectively infinite expansion
- 292/10, 5.66×10in SiC filter



# Full and Partial Particulate Filters: Nomenclature



- Label channels "inlet" or "outlet" according to the situation if the "missing" plugs were replaced

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# Filter Backpressure Models

Will compare the prediction of four different filter models

$d_i$	: Width of inlet channel
$d_o$	: Width of outlet channel
$F_i, F_o$	: Viscous loss coefficient
$P_i, P_o$	: Pressure in inlet & outlet channels
$z$	: Axial coordinate
$\alpha$	: Momentum flux correction factor
$\mu$	: Gas viscosity
$\rho$	: Gas density
$\phi_i$	: Mass flux of gas along inlet channel
$\phi_o$	: Mass flux of gas along outlet chan

- Standard model:

$$\bullet \frac{\partial P_i}{\partial z} = -\frac{\alpha_i}{d_i^2} \frac{\partial d_i^2 \phi_i^2 / \rho_i}{\partial z} - \frac{F_i \mu_i \phi_i}{d_i^2 \rho_i} \quad \frac{\partial P_o}{\partial z} = -\alpha_o \frac{\partial \phi_o^2 / \rho_o}{\partial z} - \frac{F_o \mu_o \phi_o}{d_o^2 \rho_o}$$

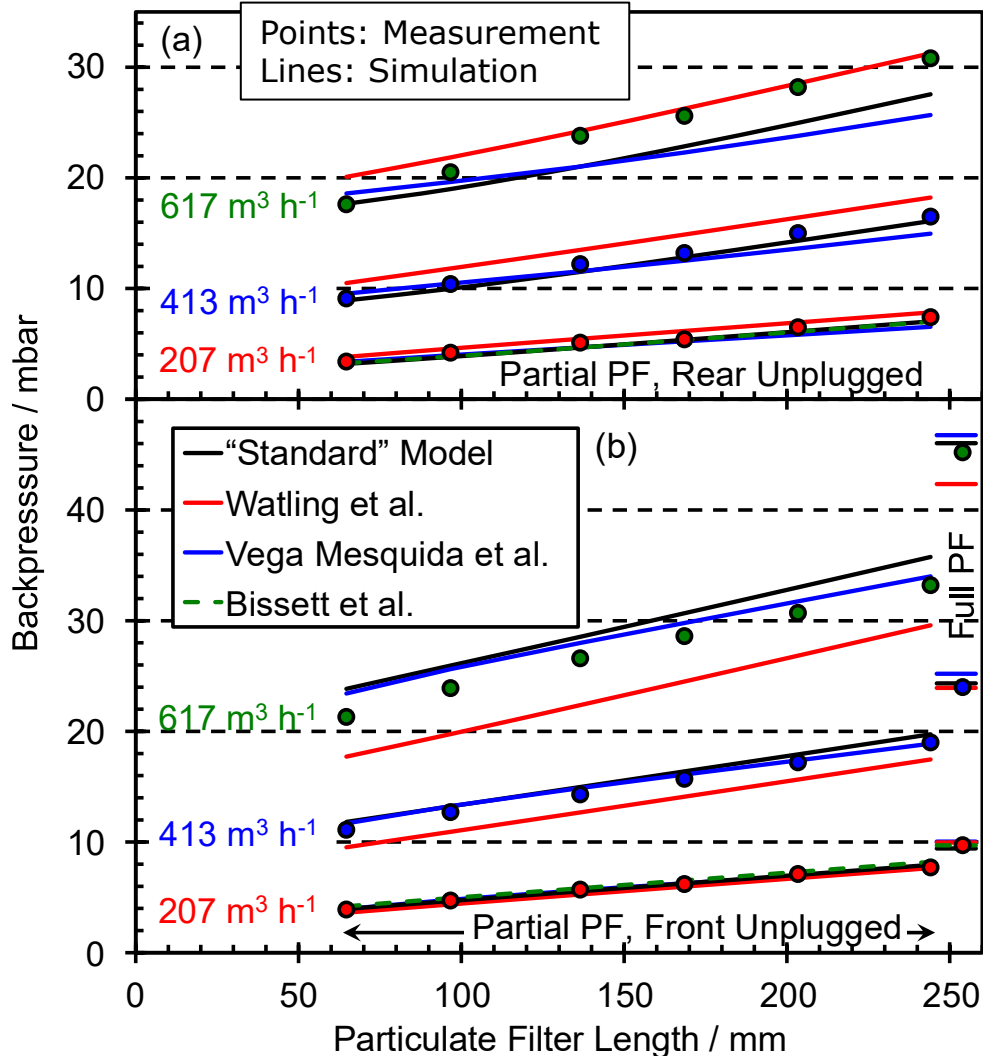
- Vega Mesquida et al. [7] model uses same equations but with different values for F &  $\alpha$
- Bissett et al. [8] model uses same equation but with  $Re_w$  dependant F &  $\alpha$
- Model proposed in Watling et al., SAE 2017-01-0974 [2]:

$$\bullet \frac{\partial P_i}{\partial z} = -\frac{\alpha_i \phi_i}{d_i^2} \frac{\partial d_i^2 \phi_i / \rho_i}{\partial z} - \frac{F_i \mu_i \phi_i}{d_i^2 \rho_i} \quad \frac{\partial P_o}{\partial z} = -\alpha_o \phi_o \frac{\partial \phi_o / \rho_o}{\partial z} - \frac{F_o \mu_o \phi_o}{d_o^2 \rho_o}$$

Equations for pressure change due flow contraction & expansion extended to cover the unplugged faces of partial filters [1]

# Comparison of Backpressure Prediction for Different Models

Permeability optimised for each model using data for two lowest flow rates



- All models look much the same at 207 m<sup>3</sup> h<sup>-1</sup>
  - Watling et al. slightly different with unplugged rear
- Bissett et al. model only valid for  $Re_W < 3$  – only true for 207 m<sup>3</sup> h<sup>-1</sup> for rear face unplugged
  - Hence, only show simulated data for 207 m<sup>3</sup> h<sup>-1</sup>
- Poor prediction for Watling et al. model
  - Backpressure higher for unplugged front face
  - This model predicts the opposite
  - Therefore this model can be rejected
- Other models look much the same at 413 m<sup>3</sup> h<sup>-1</sup>
- No model gives a good prediction at 617 m<sup>3</sup> h<sup>-1</sup>
  - Perhaps assuming developed laminar flow incorrect

# Modelling Developing Flow in Filters

## Defining a dimensionless distance for wall-flow filters

- Correlations for developing flow for monoliths in terms of dimensionless distance,  $z^+$ :

- $z^+ = \frac{z}{dRe}$

- This definition doesn't work for filters as  $Re$  varies along channels

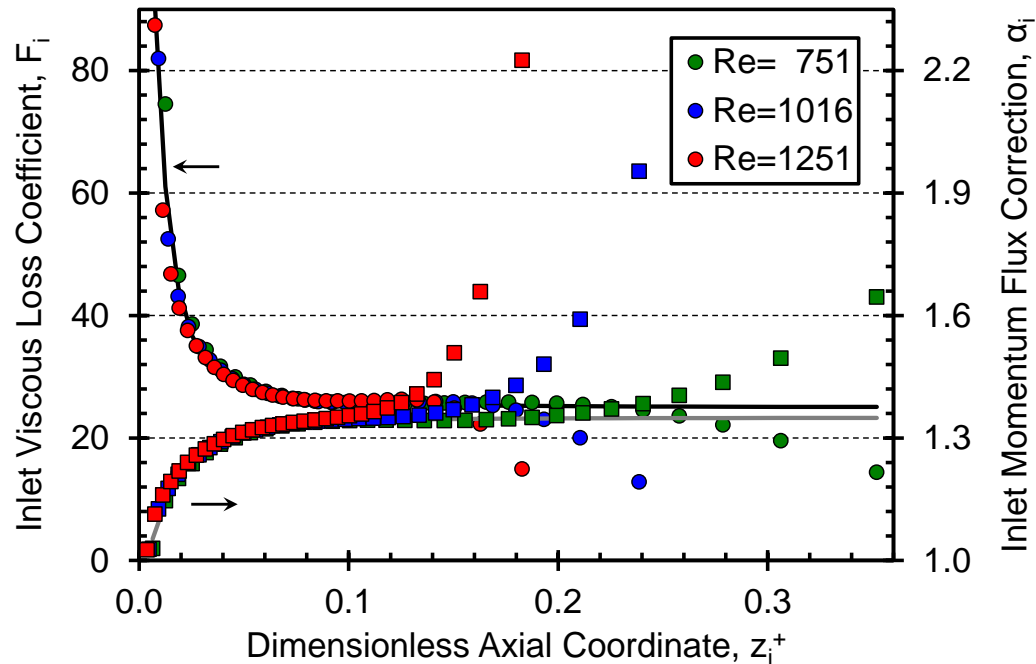
- For filters, define dimensionless distance as:

- $z_{i,l}^+ = \sum_{j=1}^l \frac{z_j - z_{j-1}}{d_i Re_{i,j}}$        $z_{o,l}^+ = \sum_{j=1}^l \frac{z_j - z_{j-1}}{d_o Re_{o,j}}$

- Divide channel up into a series of axial elements. Use values for these elements in summation

# Demonstration That Dimensionless Distance Definition Works

Use CFD results from Cooper et al. [9]



- Plot  $F_i$  and  $\alpha_i$  calculated from CFD against  $z_i^+$  for 3 values of Re
- Predicted points line up for front of filter, where impact developing flow important
  - No correlation at rear of part where wall flow is important

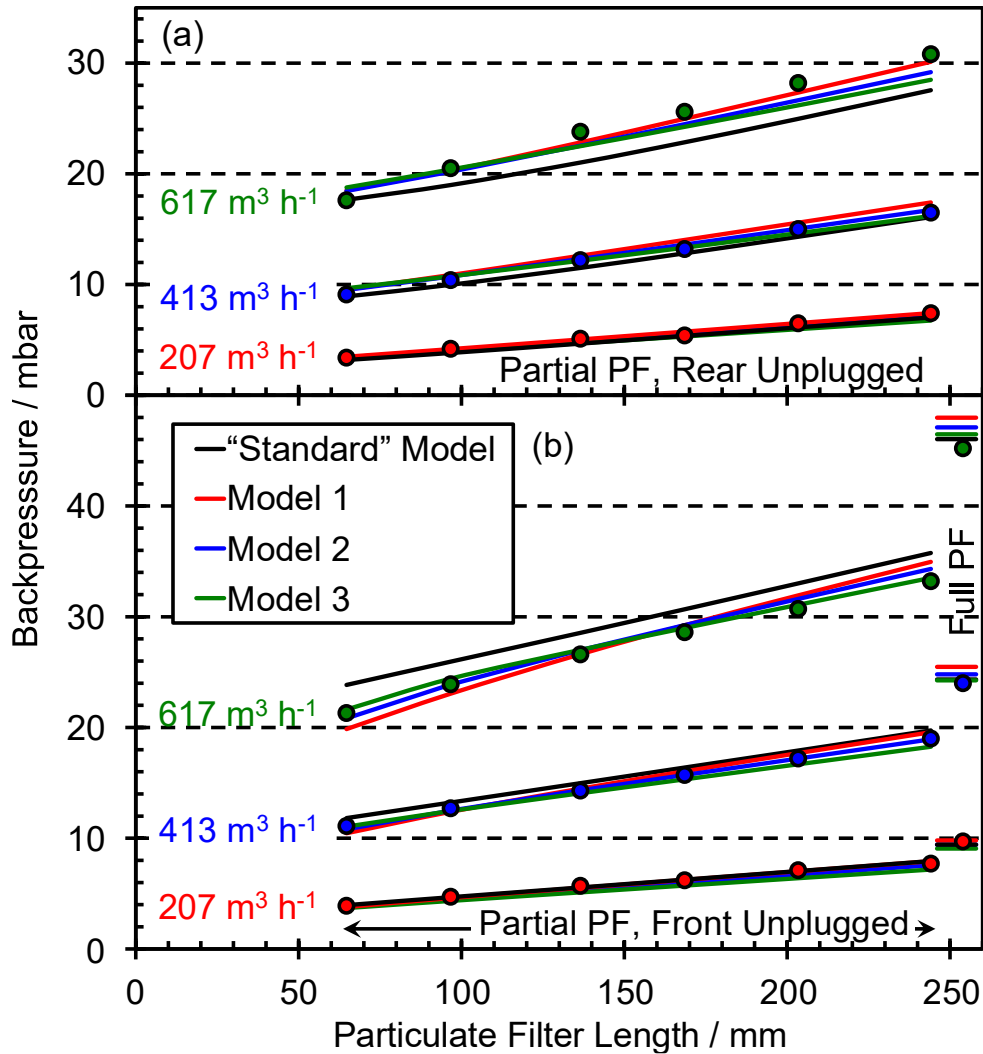
# Models Including Developing Flow

<b>Model 1</b>	$\alpha_D = 1 + \frac{0.378}{1+0.004210(z^+)^{-2}}$
<b>Model 2</b>	$\alpha_D = 1 + \frac{0.378}{1+0.001487(z^+)^{-2}}, F_i = F_o = 26.613$
<b>Model 3</b>	$\alpha_D = 1 + \frac{0.378}{1+0.0005391(z^+)^{-2}}, F_D = 27.779 - \frac{3.216}{1+0.0001720(z^+)^{-2}}$

- Equations for F and  $\alpha$  as functions of dimensionless distance
- This only applies to channels unplugged at the front face
- Form of equations taken from monolith friction factor correlation of Shah & London [10]



# Comparison of Models Including Developing Flow



- All models give much the same prediction for the two lower flows
  - Expected as impact of developing flow only significant at higher flow
- Model including developing flow show some advantage over "standard" model, but none is perfect
  - More work required
  - Perhaps there is a limit to the range of flow rates at which a 1-dimension model would work

All models use same permeability

# Summary: Cut Filter Backpressure Data

- Generate data for validating filter model by measuring backpressure after repeatedly shortening part
  - After first cut, have partial filter
  - Test part in both orientation – backpressure higher when front face unplugged
- As full and partial filters governed by same differential equations, can use data for partial filter to validate model for full filter
- Method enables model discrimination
- None of models tried works really well at really high flow
  - Have tried models including developing flow
    - This improves prediction, but prediction is far from perfect



Conclusions

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# Conclusions

- A number of aspects of backpressure prediction for Flow-Through Monoliths (FTM) & Filters (PF) investigated:
  - Equation for pressure drop due to contraction validated against CFD and experimental data
  - Good backpressure prediction for FTM by combining equation for contraction to flat velocity profile and developing flow friction factor correlation assuming flat velocity profile at channel entrance
  - Backpressure data for partial PFs can be used to validate balance equations for full PF and to discriminate between models
  - Progress made on including developing flow in PF model; more work required

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